

THERMOECONOMIC OPTIMISATION OF A HEAT PUMP SYSTEM

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Abstract. We present the application of Thermoconomics to the optimisation of a single-stage heat-pump cycle. The method is well suited for application to thermodynamic processes and yields exergy losses. The marginal cost of an arbitrary variable can also be calculated. The efficiencies of the compressor, condenser, evaporator, and electric motor are chosen as the decision variables which are to be optimised. Parameters such as the price of electricity, the temperature of the delivered heat, and the operation time may vary among optimisations, and results are presented for different parameter values.

INTRODUCTION

Contemporary technical systems are often constructed without detailed prior analysis of the effects of different potential solutions for a given system in terms of cost. Instead, constructions are often based on experience, educated guesswork and personal evaluations. Sometimes energy reallocations are made, for example when the investment budget is based on calculation of energy (or, better, exergy) losses in each zone of the system. Irrespective of the method used, the maximum cost permitted for each zone is obtained, and the market prices then determine to what extent efficient components can be afforded. Such systems always cost at least as much as and often more than they would if optimisation were used.

Tribus and El-Sayed(1, 2) developed the concept of Thermoconomics, in which the objective function is optimised, subject to given economic and technical constraints. The purpose of Thermoconomics is to improve analyses of systems by introducing ways of concurrently suggesting improvements.

The result of an insufficient analysis is that we never know the best solution or how to find it. By optimising the total system in operation, we always find the best system within the given conditions. We can also calculate the marginal costs of the exergy losses in each component. These values are very important in the selection of research and development measures, or in the improvement of an existing system.

The system is described in relation to the physical (pressure p_0 , temperature T_0 , and chemical potentials of the appropriate substances $m_{c,0}$) and economic environments (prices of goods c_i and prices of capital or interest rates a_i). These two environments are interrelated by cost relations for physical quantities.

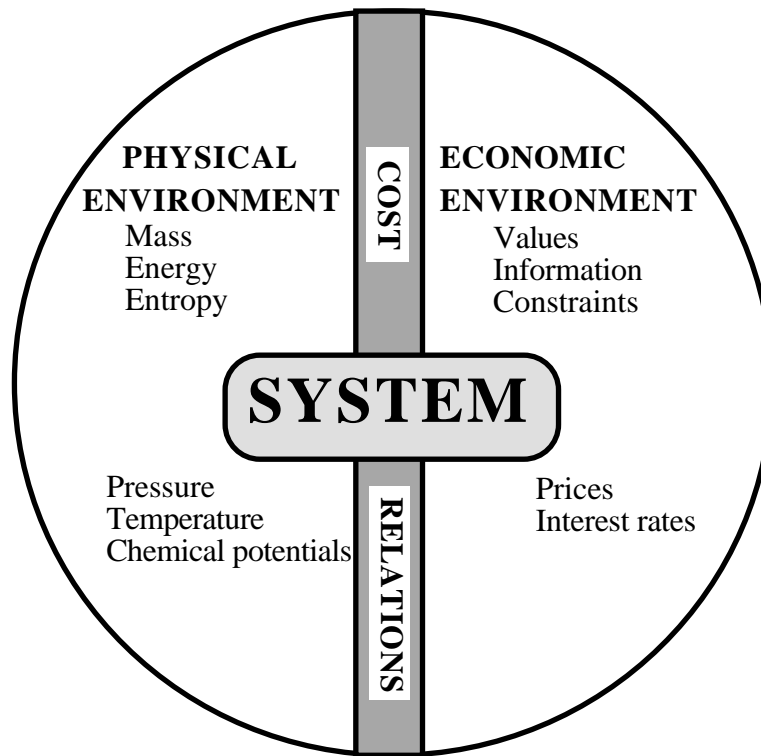


Fig. 1. The system in two environments.

Briefly, the method involves the following steps:

1. A concise description of the system under study,
2. Definition of system boundaries, zones, components, etc. (a detailed flow chart or a draft of the process),
3. Definitions of the physical environment or, alternatively, of the local physical environment,
4. Declarations of the sources of the thermophysical data,
5. Development of a computational algorithm with inputs and outputs clearly identified, based on mass and energy balances for the system and a complete thermodynamic description, within the given conditions,
6. Definitions of the cost equations and the economic objective function,
7. Determinations of the exergy flows and sources of entropy, i.e. exergy sinks, in the system (related to the in and outflows of exergy),
8. Determination of the monetary flows in the process,
9. Suggestions of possible improvements of the configuration of the system and adjustments of associated relations (5),
10. Optimisations of the objective of the process, i.e. the cost,
11. Carrying out of a sensitivity study,
12. Suggested improvements of the design and areas for further research and development.

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The initial stages in this working scheme are self-explanatory and generally accepted. The new element is introduction of the environment and its effects on the system. This makes it possible to use the exergy concept to calculate the amount of work lost due to irreversibilities in the system. Thermoeconomics find the optimum economic design within physical feasibility. All costs are based on life-time costs, which may also include maintenance costs, etc. We might not be able to realise this optimum solution for an existing system, but we can learn how to improve it. The method also provides new insights, which often lead to new, unexpected improvements. Interest in Thermoeconomics from industrial management is increasing today (3).

Thermodynamic systems can be completely described using equations of mass balance (for each substance), energy and entropy, in conjunction with thermophysical property relations and/or equations of state, equipment performance characteristics, thermokinetic or rate equations, and boundary/initial conditions. Given an adequate description of the process, it can be optimised using any current technique. The exergy flow gives a unique description of the potentials for improving efficiency.

The variables describing the system can usually be divided into basic state variables, variable decision variables, and fixed decision variables (2). These variables are all interrelated by the objective function and the equations of constraints. We might substitute the constraint equations into the objective function, leaving only the independent variables, i.e. the variable decision variables, to be optimised. However, this is not always mathematically desirable. Using Lagrange's method, we can maintain the fixed or dependent variables in the optimisation. Their shadow, i.e. marginal, prices are then given by the Lagrange multipliers.

The application of Lagrange's method to large scale systems is well known and widespread (4). Methods are also available whereby a complex thermal system is decomposed into its component parts, each component buying and selling exergy with other components (5). However, this approach requires that the problem is described in terms of exergy co-ordinates, i.e. the constraint equations representing the internal economic transactions (supply and demand equations) must be explicitly given in terms of exergy flows. The benefit of such a transformation is that the Lagrange multipliers represent prices describing the internal sales and purchases of exergy. These prices can be used in turn to show the economic trade-offs between capital investment costs and exergy losses for each component of the system.

In order to obtain the exergy balance equations, all the relevant thermodynamic constraints must be incorporated. Usually there is no general way to find these constraint equations. All state variables (temperature, pressure, mass flows, etc.) must also be transformed into exergy variables, which may be problematical. Usually this transformation ends up in a number of non-linear equations which must be solved to get the optimum values for the decision variables. We avoid these problems by using numerical treatment.

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When constructing a system, attempts should be made to attain the highest possible technical efficiency at the lowest cost within the existing technical, economical and legal constraints. (Sometimes we must also consider ethical, ecological or social consequences.) The analysis should also include different operating points (pressures, temperatures, etc.), configurations (components, flow charts, etc.), purposes (dual purpose, use of waste streams, etc.), and environments (global or local environment, new prices, etc.). As mentioned above, new areas for research and development should also be identified.

Computer programs and suitable strategies allow for examination of a large number of solutions, and determination of competitive solutions. A good strategy should include the use of new components and a suitable level of accuracy for available and assumed quantities representing different properties, performances and costs. The purpose of this study is to develop such a strategy from a fundamental theoretical basis and to formalise what many people regard as common sense. The point of departure is in regarding the system as surrounded by a physical and economic environment, see Fig. 1.

With regard to the physical environment, the energy and mass flows are evaluated in physical terms, i.e. in terms of exergy per unit time. The difference between all incoming exergy flows and all outgoing exergy flows must be minimised and the efficiency must be maximised. In the economic environment all energy and mass flows are evaluated instead in terms of economic value or costs. The main function is now the cost per unit time, (i.e. operation and capital costs minus income), which should be minimal. Thermoeconomic optimisation is economic optimisation in conjunction with thorough thermodynamic description of the system.

EXERGY

The exergy losses due to irreversibilities in a stationary state can be determined for each zone, a single component or a number of components, by regarding in and outflows of exergy. The exergy content E of a flow is

$$E = H - T_0 S - \sum_c \mu_{c,0} n_c \quad (1)$$

where H is the enthalpy, T_0 the absolute temperature of the environment, S the entropy, and $\mu_{c,0}$ the chemical potential of substance c of quantity n_c . The exergy loss can be determined for each zone. For the system, we obtain a sum for all zones, which gives the total rate of exergy loss. This may also be written as the product of the environment temperature and total rate of entropy production, i.e. $T_0 \dot{S}^{tot}$.

COSTS OF EXERGY LOSSES

There are 2 main reasons for placing a price on exergy losses, irrespective of whether the purpose is physical or economic optimisation. The first is to obtain a description of how the variable mass and energy flows in the objective function are distributed over the process

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and the second is to find a redistribution that fulfils the objectives. This corresponds to 2 kinds of analysis: *Thermoeconomics accounting* with the use of direct prices which allows comparison between zones and their costs, disregarding other parts of the system, and *Thermoeconomics optimisation* using differential prices, which allows comparison between the local and global responses of the system to a particular change of state. This leads to a sensitivity analysis and an optimisation of the defined objective function.

The differential prices indicating the direction of improving the objective function are marginal prices and shadow prices. A marginal price indicates the change of the objective function at a unit change of a variable decision variable. The shadow price indicates the corresponding direction for the other variables. Marginal prices of exergy can then be calculated using the chain rule of derivation.

EFFICIENCY AND COST OPTIMISATION

The calculation of exergy flows and their costs requires a thermodynamic and economic description of the system.[†] A suitable level of description, when comparing alternative solutions, is the level at which efficiencies and costs for each subsystem are measured (as part of the system) in common quantities. A more detailed description can thus be made by dividing the subsystem into even smaller parts.

Important relations for describing the subsystems are equations for mass, energy, performances and costs as functions of performance. Cost relations as functions of capacity parameters such as areas, mass flows and powers are suitable in a preliminary analysis (6). These relations may later be rewritten to include other parameters such as temperatures, pressures or efficiencies.

THERMOECONOMIC ACCOUNTING

This method is one based on the exergy flows in the process, by which a price is put on the exergy. The exergy inflows are shared between useful outflows, wasted outflows and exergy losses. Thermoeconomic accounting simply means determining the exergy flows and assigning values to the exergy flows and losses. When there are various in and outflows the prices may vary. If the price per exergy unit does not vary too greatly we can define an “average price”. This method allows comparison of the economic cost of a zone with the cost of its exergy loss.

Thermoeconomic accounting does not include consideration of the system effects. It does not describe how the capital investments in one part on the system affect exergy losses in other parts of the system. In the Thermoeconomic accounting method the exergy losses are only figures not functions. However, this simple type of analysis sometimes gives ideas for unusual improvements.

[†] Second law considerations are not needed in the optimization. The exergy concept is only used to discover possible future improvements beyond the economic improvements.

THERMOECONOMIC OPTIMISATION

Direct prices are not affected when a change in one part of the system affects other parts. Marginal and shadow prices are affected, but they are difficult to determine. According to the system equations they are dependent on the first derivatives of the parameters in question and not only on their absolute values.

The objective function ϕ_0 must be defined as a function of dependent state variables or state parameters $\{x_j\}$, where $\{x_j\}$ is abbreviation for $x_1, x_2, \dots, x_j, \dots, x_n$, variable decision variables or just decision variables $\{y_k\}$, and fixed decision variables or decision parameters $\{z_l\}$, i.e.

$$\phi_0 = \phi_0(\{x_j\}, \{y_k\}, \{z_l\}), \quad (2)$$

where $j=1, 2, \dots, n$, $k=1, 2, \dots, m$, and $l=1, 2, \dots, r$.

The equations of state may be divided into 2 groups. One group fixes the dependent variables $\{x_j\}$, appearing in the objective function ϕ_0 and in the constraints, $\{g_j = 0\}$, in Lagrange's method. The other group is related to the Lagrange multipliers $\{\lambda_j\}$. The equations of state are written as

$$g_j = g_j - x_j, \quad (3)$$

where g_j is the function used in Lagrange's method since it must be zero at optimum, i.e. when $x_j = g_j$. g_j gives the relations between the state variable in question x_j and the other state variables $\{x_i\}_{i \neq j}$ and the decision variables $\{y_k\}$ and $\{z_l\}$. With these definitions, the equations of state become

$$g_j(\{x_i\}, \{y_k\}, \{z_l\}) = 0, \quad j = 1, 2, \dots, n. \quad (4)$$

The problem can now be formulated as follows:

$$\text{minimise } \phi_0 = \phi_0(\{x_i\}, \{y_k\}, \{z_l\}), \quad (5)$$

$$\text{subject to } g_j(\{x_i\}, \{y_k\}, \{z_l\}) = 0, \quad j = 1, 2, \dots, n. \quad (6)$$

With the defined objective function and equations of state, the Lagrangian becomes

$$L = \phi_0 + \sum_j \lambda_j g_j. \quad (7)$$

According to Lagrange's method,

$$L/x_i = 0, \quad i = 1, 2, \dots, n. \quad (8)$$

This procedure gives n equations, which are linearly dependent through the unknown Lagrange multipliers $\{\lambda_j\}$, which may be regarded as shadow prices (7).

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When the objective function and the equations of state are not too complicated, the optimum is given by

$$L/ y_k = 0, \quad k=1, 2, \dots, m. \quad (9)$$

The optimum solution is found by solving these equations for the variable decision variables $\{y_k\}$.

In real world applications, it is often better to define a marginal price θ_k , related to each variable decision variable y_k , i.e.

$$\mu_{c0} n_c$$

$$\theta_k = \frac{L}{y_k} = \frac{0}{y_k} + \sum_{i=1}^n \lambda_i \frac{i}{y_k}, \quad k = 1, 2, \dots, m. \quad (10)$$

The physical interpretation of these prices is that they constitute the gradients for each variable decision variable, y_k , i.e. they give the direction for improving the system.

Equation (8) represents n linear equations for the unknown Lagrange multipliers and Eq. (9) represents m equations with the variable decision variables as unknowns. In many real world applications of this method the $n+m$ equations defined by Eqs. (8) and (9) are strongly non-linear and difficult to solve. Numerical treatment is therefore often necessary. Numerical values must then be calculated for the shadow prices $\{\lambda_j\}$ for every set of the variable decision variables $\{y_k\}$. These shadow prices are then inserted into Eq. (10) to determine the marginal prices $\{\theta_k\}$. The marginal prices represent the derivatives of the objective function L_0 in the state i . From the values of the marginal prices we can now indicate the direction for improving the system. A new state, $i+1$, is chosen in this direction according to

$$y_{k,i+1} = y_{k,i} \pm \theta_k \quad (11)$$

where θ_k is a predefined step.

When the marginal price $\theta_{k,i}$ is negative then the derivative of the objective function is negative. If we want to minimise the objective function, we must increase the variable decision variable $y_{k,i}$ to reach the optimum. If the marginal price is positive we must decrease the variable decision variable.

SYSTEM MODEL

Heat pump systems offer much more efficient means of producing heat than traditional combustion or electrical short circuit technologies. Heat pump systems are therefore becoming more common as the prices of fuels and electricity increase. The configuration of the system is defined in Fig. 2. It consists of a compressor, a condenser, an expansion valve, an evaporator, and an electric motor.

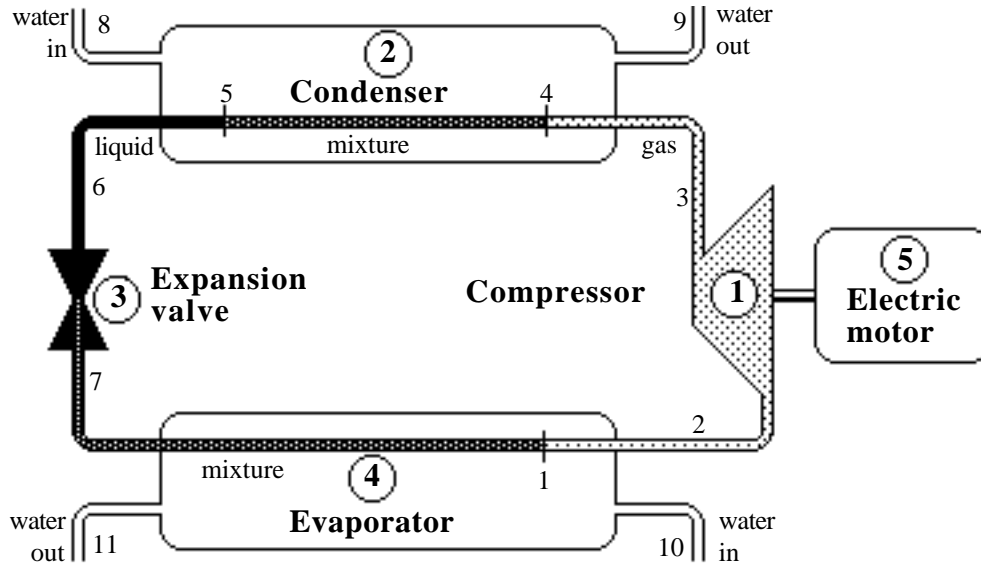


Fig. 2. Heat pump system with 5 components and 11 flows.

The refrigerant is superheated after passing through the evaporator, step 1-2, and super cooled after passing through the condenser, step 5-6. The actual state of the refrigerant after the compression, 3, differs from that of a reversible process, 3_{rev} , due to the limited efficiency of the compressor. The heat produced from the system is $h_3 - h_6$, the heat input is $h_2 - h_7$, and the work supplied to the compressor is $h_3 - h_2$. The electricity input required to operate the system becomes $(h_3 - h_2)/\eta_5$, where η_5 is the efficiency of the electric motor.

Variable decision variables are the efficiencies of the compressor, the condenser, the evaporator, and the electric motor. These are defined as follows

$$\eta_1 = \frac{h_{3rev} - h_2}{h_3 - h_2}, \quad (12)$$

$$\eta_2 = \frac{T_9 - T_8}{T_4 - T_8}, \quad (13)$$

$$\eta_4 = \frac{T_{11} - T_{10}}{T_1 - T_{10}}, \quad (14)$$

$$\eta_5 = \frac{m_r (h_3 - h_2)}{P} \quad (15)$$

where m_r is the mass flow of the refrigerant, and P is the electric power.

The system is completely defined apart from the variable decision variables $\{y_k\}$, each set of which determines a state of the system. The exergy flows and exergy losses are also determined for each component.

The objective is to minimise the cost for a given amount of produced heat. The cost includes both the operating (electricity) cost and the capital cost. The operating cost increases if the investments decrease and vice versa. The income from the product (heat) and a given required value of the profit sets an upper limit for the total cost of the system.

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The problem is to split this cost between the operating cost and the capital cost for each component. (The costs for parts not affected by alternative constructions of the system, such as pipes connecting the components, are just added as constants and have no effect on the optimisation.) The costs, C , for each of the 5 components will now be given.

$$\text{Compressor} \quad C_1 = a_1 k_1 \frac{V_2}{0.9-\eta_1} \frac{p_3}{p_2} \ln \frac{p_3}{p_2} \quad (16)$$

where k_1 is the cost per volume flow, V_2 , p_3/p_2 is the pressure ratio over the compressor, and η_1 is the enthalpy efficiency.

$$\text{Condenser} \quad C_2 = a_2 k_2 m_{wh} \sqrt{\frac{\eta_2}{1-\eta_2}} = a_2 k_2 m_{wh} \sqrt{e^{NTU_2} - 1}, \quad (17)$$

where k_2 is the cost per mass flow of water, m_{wh} , on the hot side, η_2 is the temperature efficiency, and NTU_2 is the number of heat transfer units.

$$\text{Expansion Valve} \quad C_3 = a_3 k_3 m_r, \quad (18)$$

where k_3 is the cost per mass flow of the refrigerant.

$$\text{Evaporator} \quad C_4 = a_4 k_4 m_{wc} \sqrt{\frac{\eta_4}{1-\eta_4}} = a_4 k_4 m_{wc} \sqrt{e^{NTU_4} - 1}, \quad (19)$$

where k_4 , η_4 and NTU_4 are analogous to the condenser above and m_{wc} is the mass flow of water on the cold side.

$$\text{Electric Motor} \quad C_5 = a_5 k_5 P \frac{\eta_5}{1-\eta_5}, \quad (20)$$

where k_5 is the cost per unit power and η_5 is the efficiency. Furthermore a_1 - a_5 are annuity factors of the different capital investments defined as

$$a_i = \frac{r}{1 - (1+r)^{-n_i}}. \quad (21)$$

where r is the interest rate and n_i is the depreciation time for component i . The depreciation time may vary for each component due to variations in economic lifetime and maintenance costs such as renovations, etc.

Figure 3 shows the investment costs as a function of the efficiencies. The values of the fixed decision variables are set, for instance as follows: heat produced 6500 W (energy-power), operation time/yr 5000 hr, price of electricity 0.25 SEK/kWh, and temperature of the produced heat (T_9) 60°C.

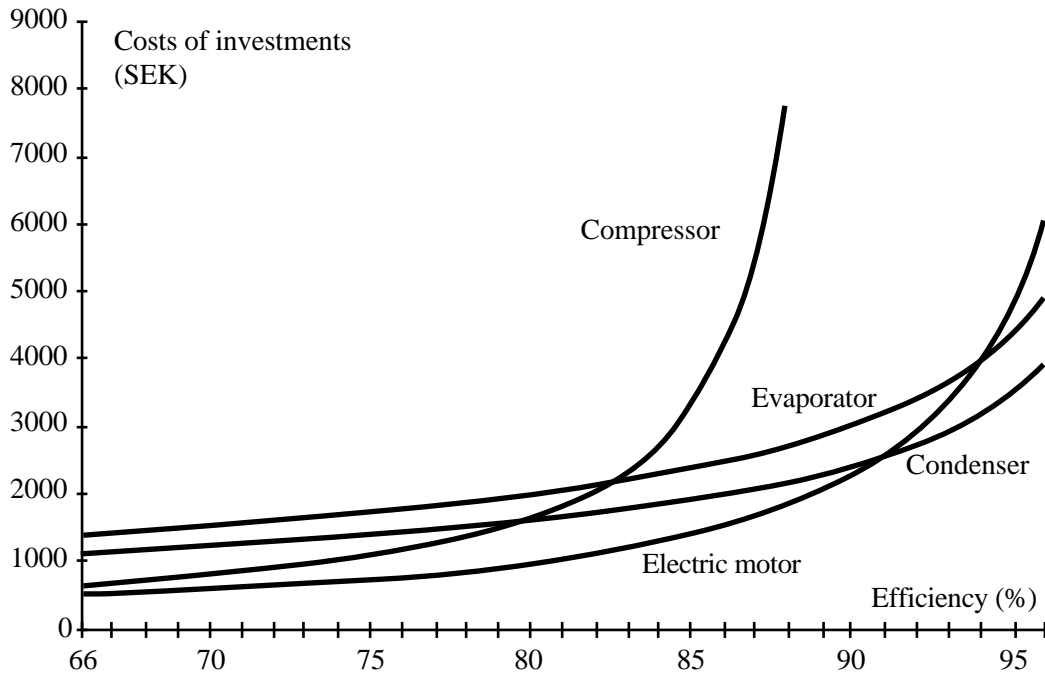


Fig. 3. Costs of investments as a function of the efficiencies.

The investment costs are depreciated according to the annuity method, which gives a cost per unit time for every component. The total cost per unit time \dot{C}_0 (= objective function) is the sum of these costs and the cost of the electricity used, E_{el} , i.e.

$$\dot{C}_0 = \sum_{i=1}^5 C_i + t p_{el} E_{el}. \quad (22)$$

where t is the operating time per unit time and p_{el} is the price of electricity.

RESULTS AND CONCLUSIONS

This optimisation problem is too complicated to be solved by Lagrange's method. Instead, the optimisation is achieved by calculating the value of the objective function \dot{C}_0 and the marginal costs $\{\theta_k\}$ for every set of the variable decision variables $\{y_k\}$ according to

$$\theta_k = \dot{C}_0 / y_k, \quad k = 1, 2, 3, 4. \quad (23)$$

From these values a new set of variable decision variables $\{y_k\}$ is determined by using the Newton-Raphson method (8). Thus the system moves towards the nearest minimum from the given start values. However, the problem is strongly non-linear which means that there is no general method for finding the global minimum. Instead common sense and insight into how the system works must be used to determine the value of a solution.

A computer program has been developed for finding the optimum system. The program begins by calculating the thermodynamic data for the assumed refrigerant R12. (Other refrigerants may also be used.) These calculations are based on similar computer-based

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calculations by Reynolds (9). The equations of state for the system are formulated so as to avoid iterations. The actual minimising procedure is carried out with a small number of iterations. When the sum of the marginal prices is less than a predefined value the optimisation is completed.

Let us assume a system with the values 0.7, for all efficiencies, which roughly correspond to real values of a system of this size, the variable decision variables, which are to be optimised. The calculated total cost then becomes SEK 4221/yr of which SEK 3617/yr relate to electricity. The optimisation gives the following efficiency values: compressor 0.80, condenser 0.83, evaporator 0.73 and electric motor 0.91, the total cost now amounting to SEK 3388/yr of which SEK 2416/yr is for the electricity. By increasing the investments from SEK 604/yr to SEK 972/yr, the total cost of the system becomes SEK 833/yr less than for the assumed system, see Fig. 4. At the same time the exergy losses decrease from 1933 W to 979 W, i.e. 954 W! From Fig. 5 we see that it is the improvement of the electric motor that gives the largest single exergy saving. The optimisation saves both money and exergy.

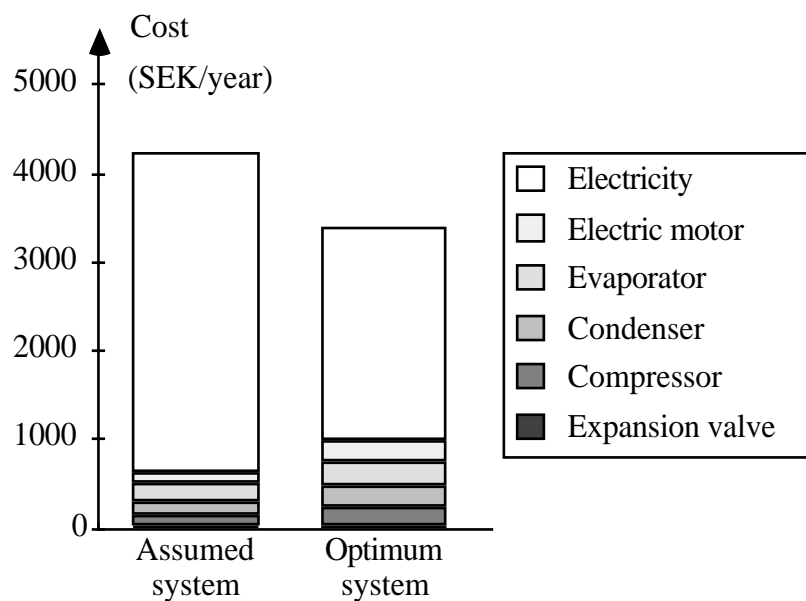


Fig. 4. Costs for the assumed and the optimum systems.

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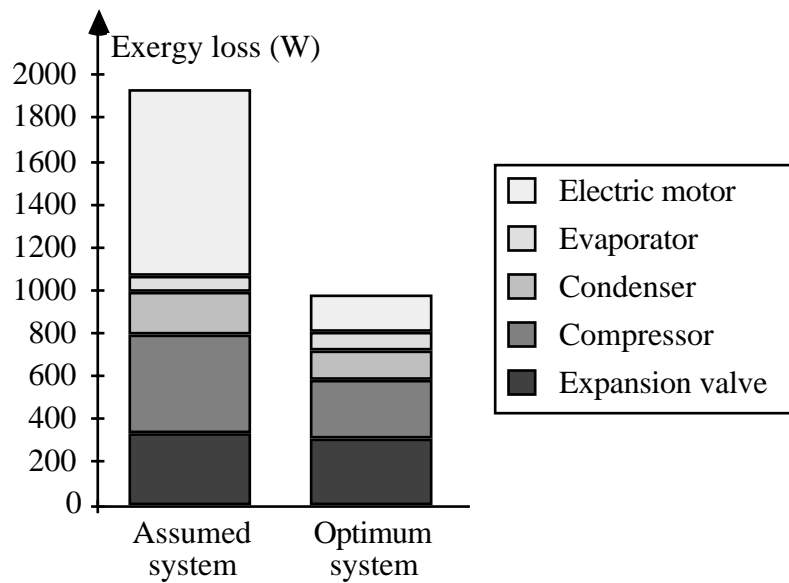


Fig. 5. Exergy losses for the assumed and the optimum systems.

The expansion valve accounts for the largest exergy loss in the optimum system, which justifies investment in research and development to improve it (10).

The result indicates the electric motor to be the most critical component to improve. The electric motor is assumed to cost approximately 3 times as much at 91% efficiency than at 70% efficiency, which must be regarded as realistic. (It may even cost up to 9 times as much and still be competitive with the assumed system.) It may also be added that the coefficient of performance (COP) increases from 2.25 for the assumed system to 3.36 for the optimal system.

The dependence on or sensitivity to the fixed decision variables (price of electricity, temperature of the produced heat, and operation time) has also been studied. Figure 6 shows the relationship between the costs and the price of electricity, when this varies between SEK 0.15 and 0.6/kWh. The total cost then increases from SEK 2073/yr to SEK 5522/yr at SEK 0.6/kWh. This can be seen in relation to the fact that if the optimum system at SEK 0.15/kWh had been used at SEK 0.6/kWh, then the total cost would be SEK 5947, i.e. an increase in the cost of SEK 425/yr. (For the assumed system the total cost would instead be SEK 9284, i.e. a cost increase of SEK 3762/yr.) Figure 7 shows more clearly the relationship between component costs and the price of electricity. All components should become more efficient (and therefore more expensive) when the price of electricity increases. This might have been anticipated, but the exact interrelations could not have been hypothesised.

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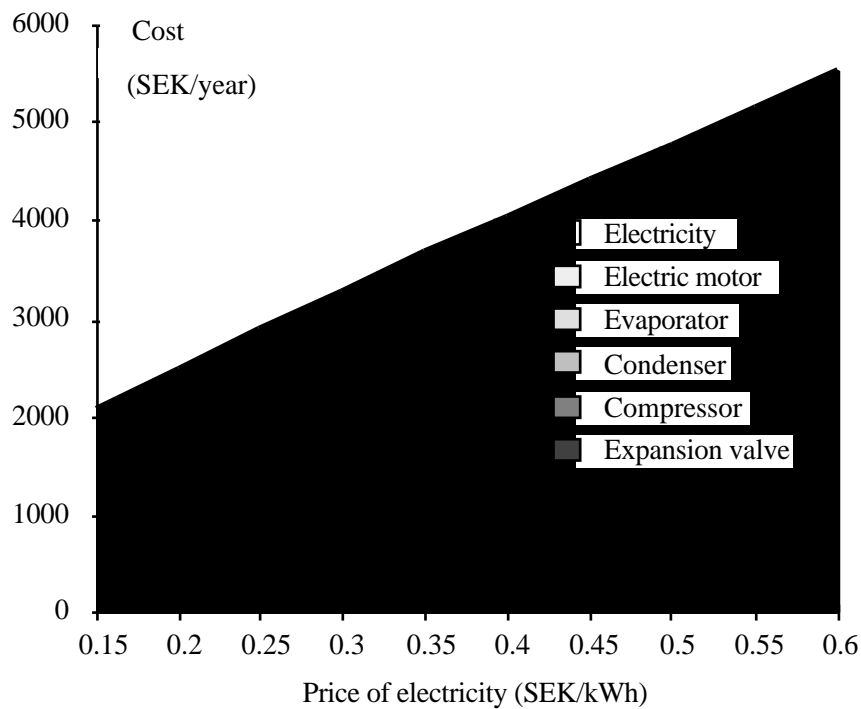


Fig. 6. Costs as a function of the price of electricity.

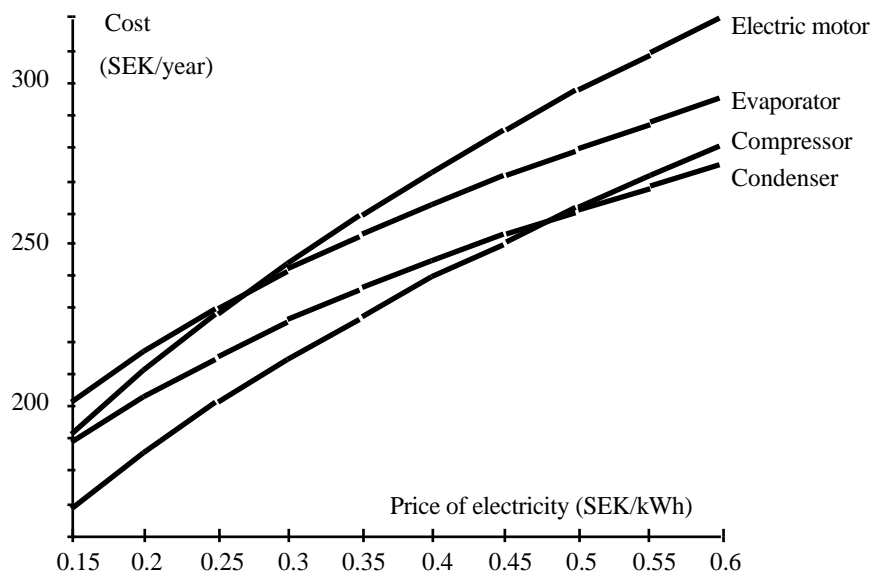


Fig. 7. Component costs as a function of the price of electricity.

When the temperature of the condenser, i.e. the temperature of the produced heat, is changed interesting things occur (Figs 8 and 9). The total cost doubles from 2336 at 40°C to SEK 4680/yr at 75°C. The heat produced in energy per time units is the same, i.e. 6500 W, but in exergy per time units it changes from 434 to 713 W which better explains the increased cost. When the temperature increases from 40 to 75°C the total system, but not necessarily each component, must be more efficient see Fig. 9. Within a total increase of

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component costs and efficiencies, it is more economical to choose a less expensive evaporator. The explanation is simply that investments in other parts of the system pay off better.

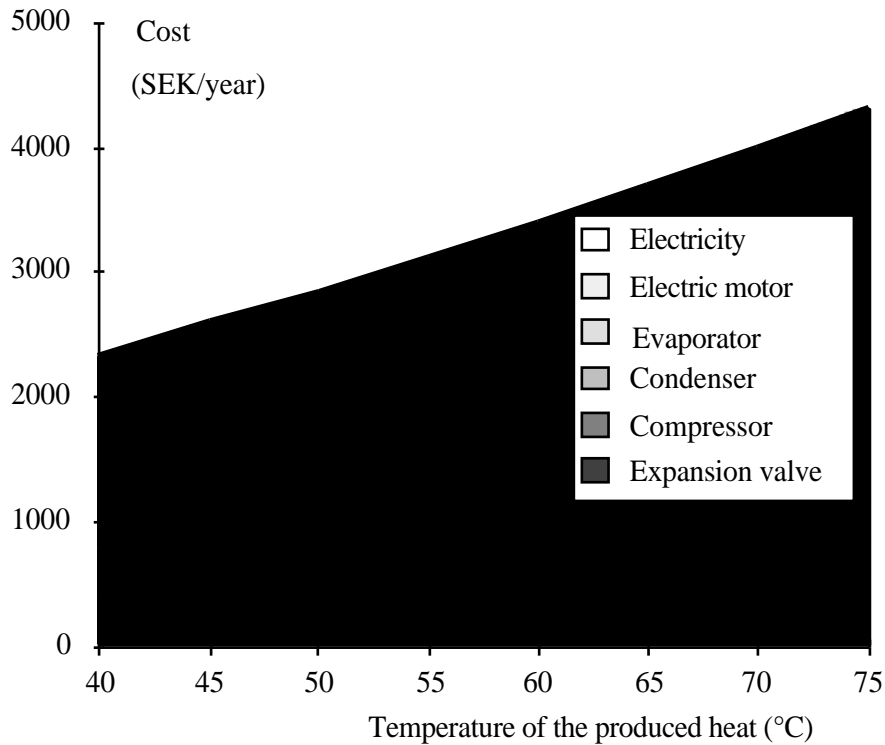


Fig. 8. Costs as a function of the temperature of the produced heat.

The system is very sensitive to the operating time which is shown in Figs 10 and 11.

Similarly, all other relations may be described using the computer program for the system. The program can easily be rewritten for other refrigerants or cost relations. The purpose of this study is merely to show the gain achieved by applying the method of Thermoeconomics to a heat pump process. The exact numerical results for describing Thermoeconomics as a method for improving technical processes have therefore been neglected.

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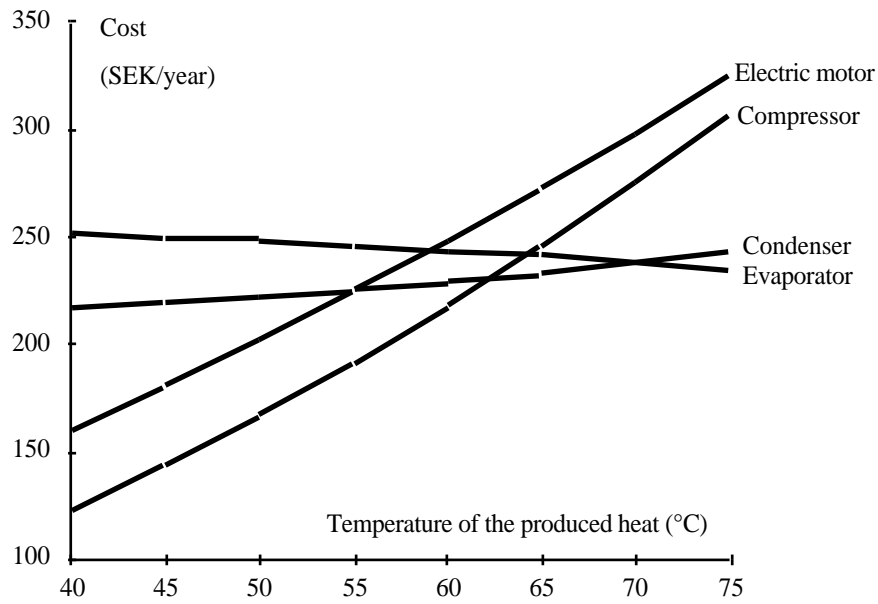


Fig. 9. Component costs as a function of the temperature of the produced heat.

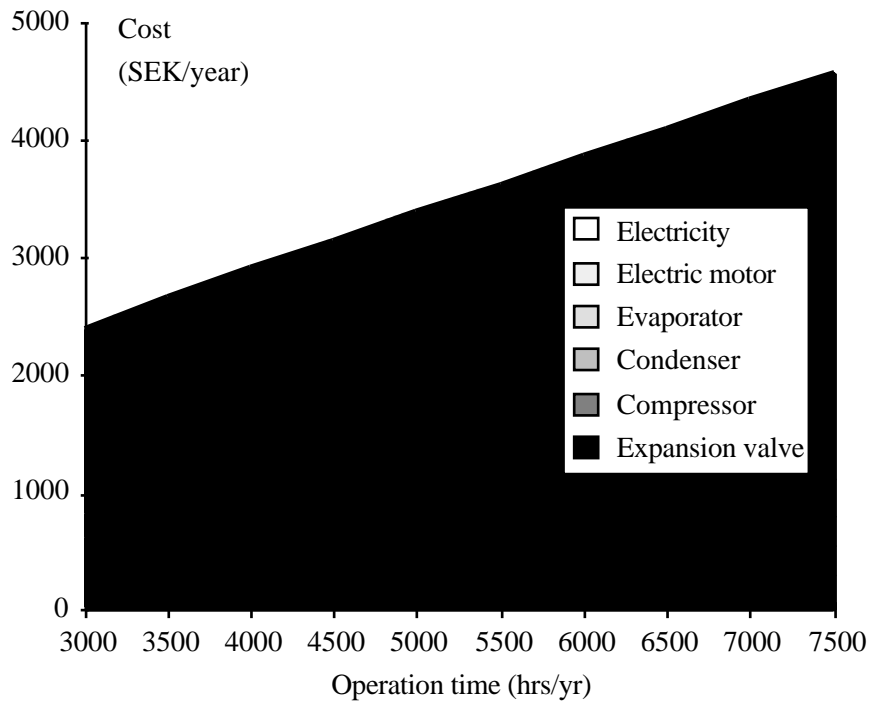


Fig. 10. Costs as a function of the operating time.

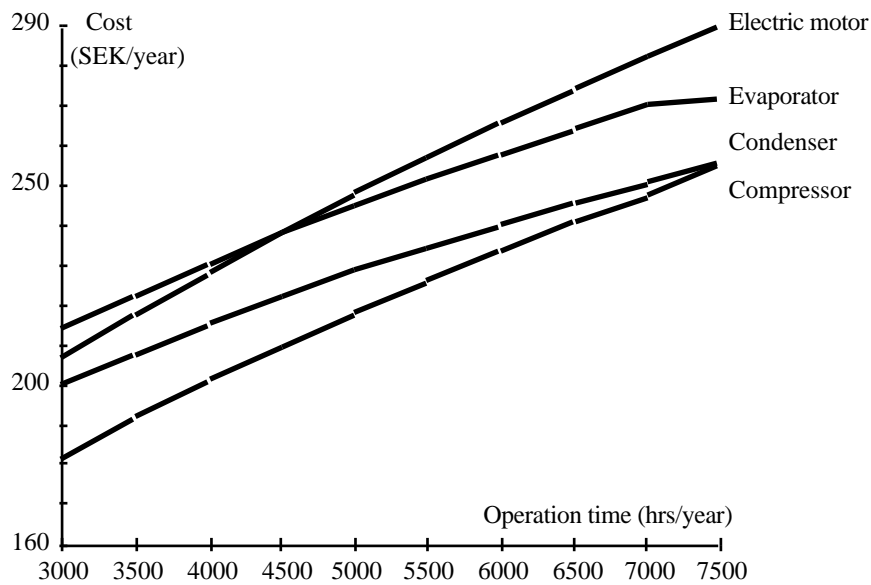
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Fig. 11. Component costs as a function of the operating time.

Thermoeconomics can never replace long experience and high technical and economic competence, but it might be an important complementary tool.

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Note — All computer programs are listed below in a separate paper.

REFERENCES

1. Y. M. El-Sayed and M. Tribus, "A Specific Strategy for the Improvement of Process Economics", Center for Advanced Engineering Study, M.I.T., Cambridge, MA, U.S.A., 1982.
2. Y. M. El-Sayed and M. Tribus, "Strategic use of Thermoeconomics for systems improvement", in R. A. Gaggioli ed., *Efficiency and Costing*, ACS Symp. ser. No. 235, 1983.
3. A. Reis, I. Smith, K. Stephan, and J. L. Peube eds., *Energy economics and management in industry*, Proc. Eur. Cong., Algarve, Portugal, 2-5 April 1984, Pergamon Press, 1985.
4. R. M. Garceau and W. J. Wepfer, "Thermoeconomic optimization of a rankine cycle cogeneration system", in R. A. Gaggioli ed., *Efficiency and Costing*, ACS Symp. ser. No. 235, 1983.
5. Y. M. El-Sayed and R. B. Evans, *J. Engng Power*, vol. 27, January, 1970.

Thermodynamic optimization of a Heat Pump System

6. K. K. Humphreys and S. Katell, *Basic Cost Engineering*, Marcel Dekker, New York, 1981.
7. Y. M. El-Sayed and M. Tribus, "The strategic use of Thermoeconomic analysis for process improvement", Pres. at the AIChE Meet., Detroit, August, 1981.
8. R. L. Johnston, *Numerical Methods a Software Approach*, Wiley, New York, 1982.
9. W. C. Reynolds, "Thermodynamic properties in SI - Graphs, Tables and Computational Equations for 40 Substances", Dep. Mech. Eng., Stanford Univ., CA 94305, U.S.A., 1979.
10. E. Granryd, Res. proj. no. 80-4062 supported by the National Swedish Board for Technical Development, Box 432 00, S-100 72 Stockholm, Sweden, 1985.